Amendments to the Specification

Please replace the paragraph beginning at page 6, line 9, with the following rewritten paragraph:

The upper part of the nut body inclusive of the first and second push parts is circular in plan view shape. Thus, the first and second push parts are free from being hooked by any tool such as a spanner, and are not deformed during work. The nut body part which is located below the part which is circular in plan view, usually hexagonal in plan view shape, but it may be quadrangular, octagonal, circular, etc. in ~~pan~~ plan view shape as well. In the case that the loosening-proof nut according to the present invention, in which the slits are formed symmetrically, the dimensions according to the third to sixth aspects of the present invention are applicable to the height h of the nut body, bottom width g of the slits and the height ~~h~~ a of the push parts.

Please replace the paragraph beginning at page 9, line 24, with the following rewritten paragraph:

The outer periphery of the second axial part is circular in shape. The first and second axial part have substantially the same shape. The female thread part formation surface direction of the second axial part is set to be outward from the axis of the nut. As with the pair of slits, a plurality of slit pairs are formed at predetermined positions uniformly subtending the circumference. The maximum outer diameter of the second axial part is smaller than the minimum outer diameter of

the first axial part. The outer periphery of the second axial part is circular in shape.

Please add the following new paragraphs after the paragraph ending at page 10, line 21:

Figs. 5(A) and 5(B) are a plan view and a sectional view, respectively, showing a loosening-proof nut with more than one slit pair.

Fig. 6 is an exemplary quadrangular loosening-proof nut consistent with the present invention.

Fig. 7 is an exemplary loosening-proof nut with slits spaced apart by 120 degrees.

Please replace the paragraph beginning at page 11, line 26, with the following rewritten paragraph:

The opposite sides of the nut body 11 are formed with a first and a second diametrically symmetrical slit 13 and 14, thus forming a first and a second corresponding push part (or loosening-proof part) 15 and 16 of the nut body 11. The first and second slits 13 and 14 have to be formed in parts of the female thread 12. In this embodiment, the first and second slits 13 and 14 are formed symmetrically with respect to and at right angles (90 degrees) to the axis of the female thread 12. The first and second slits 13 and 14 may also be formed to be at an angle between 70 and 90 degrees to the axis of the female ~~screw 2~~ thread 12. While in this embodiment the nut body 11 is formed on the opposite

sides thereof with two slits 13 and 14 on the opposite side, it is also possible to form three 120 degrees spaced-apart slits reaching the inner female thread 12.

Please replace the paragraph beginning at page 14, line 5, with the following rewritten paragraph:

For the above reason, the distance b between the bottoms of the first and second slits 13 and 14 is set to 0.15 to 0.8 times (preferably 0.3 to 0.7 times) the nominal diameter d. When the distance b is larger than this range, no region for the formation of the partial threads 21 and 22 is provided. When the distance d is smaller than this range, on the other hand, the mechanical strength of parts 27 and 28 between the bottoms of the first ~~ad~~ and second slits 13 and 14 is reduced. Thus, it is necessary to set the repulsive force F1 of the partial female threads 21 and 22 when the bolt is screwed through the loosening-proof nut 10 to be less than a mechanical strength F2 not exceeding the elastic limit of the parts 27 and 28.

Please replace the paragraph beginning at page 14, line 19, with the following rewritten paragraph:

The bottom width (or bottom height) g of the first and second slits 13 and 14 is set to be 0.05 to 0.2 ~~ties~~ times (preferably 0.08 to 0.15 times) the nominal diameter d. This is so because when the bottom width g is less than the nominal diameter d, no bending margin of the first and second push parts 15 and 16 is provided. When the bottom with g exceeds 0.2 times, on the other

hand, the first and second push parts 15 and 16 are excessively bent, and also the regions of the partial female threads 21 and 22 and the female thread 12 are reduced.

Please replace the paragraph beginning at page 15, line 13, with the following rewritten paragraph:

When the bolt is screwed through the loosening-proof nut 10, the downwardly bent first and second push parts 15 and 16 are raised. Thus, the maximum load is produced in the stem parts 23 to 26 of the first and second push parts 15 and 16. Thus, the loosening-proof nut 10 can be used repeatedly so long in a range that these parts do not undergo plastic deformation. The load for providing the loosening-proof action with respect to the bolt, is produced in the part, in which the loosening-proof threads 21 and 22 are screwed through the bolt. In this embodiment of the loosening-proof nut 10, when a predetermined bolt is screwed through the loosening-proof nut 10 in the state that the tips of the first and second push parts 15 and 16 of the nut 10 are pushed down (that is, with slit width s of zero or nearly zero), for the first time the first and second push parts 15 and 16 are raised with plastic deformation, and in the state that the screwed state of the bolt is released, the first and second push parts 15 and 16 are slightly raised to increase the slit width s. For the second and further times, however, the first and second push parts 15 and 16 do not undergo additional plastic deformation, and thus the loosening-proof nut 10 can be used repeatedly. The

screwing of the bolt through the loosening-proof nut 10 requires a predetermined preparing torque.

Please replace the paragraph beginning at page 16, line 9, with the following rewritten paragraph:

This load (i.e., loosening-proof load) depends on the broadness and bending extent of the partial female threads 21 and 22 screwed on the male thread. Also, it is sufficient that the height of the loosening-~~roof~~ proof nut 10 and the dimensions of the first and second slits 13 and 14 and the first and second push part 15 and 16 satisfy the above dimensions and that the area S1 of the partial female threads 21 and 22 is 0.04 to 0.3 times (preferably 0.08 to 0.25 times) the substantial female thread area (πdh) of the loosening-proof nut 10. When the area S1 of the partial female threads 21 and 22 is smaller than this range, the loosening-proof load is reduced. When the area S1 is larger than the range, on the other hand, the tightening force of the loosening-proof nut itself is insufficient. The above dimensions and areas are also applied to a second embodiment of the loosening-proof unit 30 according to the present invention to be described hereinunder.

Please add the following new paragraphs after the paragraph ending at page 20, line 19:

Figures 5(A) and 5(B) illustrate an exemplary loosening-proof nut 50, in accordance with the present invention. Figure 5A shows exemplary nut 50 in plan view. Nut 50 includes a central female thread 512 having a

nominal diameter d. Nut 50 includes a plurality of slit pairs, first slit pair (513, 514), second slit pair (513', 514') and third slit pair (513'', 514'') formed at predetermined positions uniformly subtending the circumference. Nut 50 includes the 3 slit pairs: (513/514), (513'/514'), (513''/514'') and corresponding downwardly bent push part pairs (515/516), (515'/516'), (515''/516''). The cross sectional view of Figure 5B shows that the push parts have been bent downward. Each of the slits (513, 514, 513', 514', 513'', 514'') are formed to be symmetrical with respect to the axis of the nut, radially penetrate the female thread 512 from the outer periphery of the nut 50 and are located at an axial position (one height) on the upper side of the axial center position of the nut body 511. Each of the slits (513, 514, 513', 514', 513'', 514'') in the nut 50 penetrates the female thread 512 from the outer periphery and the bottom of the slits (513, 514, 513', 514', 513'', 514'') are circular, e.g., cut from a circular blade penetrating the nut.

Figure 6 illustrates, an exemplary quadrangular, e.g., square, loosening-proof nut 60, in accordance with the present invention, and is shown in plan view. Nut 60 includes 2 slit pairs (613/614), (613'/614') and corresponding downwardly bent push part pairs (615/616), (615'/616'). Each of the slits (613, 614, 613', 614') are formed to be symmetrical with respect to the axis of the nut and radially penetrate the female thread 612 from the outer periphery of the nut 60. The slits (613, 614, 613', 614') are located in the upper half of the nut, and the push parts, located above the slits are downwardly bend.

The slits (613, 614, 613', 614') are cut at an axial position (same height) in the nut.

Figure 7 illustrates another exemplary hexagonal nut 70 and is shown in plan view. In nut 70, three 120 degree spaced apart slits reach the inner female thread 712. Nut 70 includes the 3 slits (713, 713', 713'') and 3 corresponding push parts (715, 715', 715''). The push parts are downwardly bent. The slits (713, 713', 713'') are located in the upper half of the nut and the push parts (715, 715', 715'') are located above the slits and are downwardly bent. Each of the slits (713, 713', 713'') are cut at an axial position (same height) in the nut.